Proceedings of the 2nd World Congress on Mechanical, Chemical, and Material Engineering (MCM'16) Budapest, Hungary – August 22 – 23, 2016 Paper No. HTFF 144 DOI: 10.11159/htff16.144

Evaporation Heat Transfer Coefficients Of R-446A And R-1234ze(E)

Minsoo Kim¹ and Keumnam Cho^{2*}

¹Graduate School of Mechanical Engineering, Sungkyunkwan University Seobu-Ro, 2066, Jangan-gu, Suwon, Korea, starsteam1@skku.edu ^{2*}School of Mechanical Engineering, Sungkyunkwan University Seobu-Ro, 2066, Jangan-gu, Suwon, Korea, keumnamcho@skku.edu

Abstract - The present study investigated evaporation heat transfer coefficients and pressure drops of R-446A and R-1234ze(E). The heat transfer coefficients were investigated by measuring wall temperature on a straight stainless tube with an inner diameter of 7.4mm and a length of 1.3m and refrigerant temperature estimated by using refrigerant pressure. The test apparatus was consisted of an evaporator tube, a pump, a constant temperature bath, a pre-heater, a condenser, a DC power supply, measuring devices, etc. Sightglasses were installed at the inlet and outlet of test section to monitor flow regime. The heat transfer coefficients were measured for the quality from 0.1 to 0.9, the mass flux from 100 to 400 kg/m²s, the heat flux of 10 to 30 kW/m², and the saturation temperature from 5 to 10°C. The effect of mass flux on the evaporation heat transfer coefficient was not much at low quality, while it was relatively large at high quality. The heat transfer coefficients of R-446A were mostly larger by approximately 16% than those of R-410A, while heat transfer coefficients of R-1234ze(E) were about 3% lower than those of R134a. Pressure drops of R-446A were larger by approximately 30% than those of R-410A, while pressure drops of R-446A with the deviation of 45% and those of R-1234ze with the deviation of 26%. Pressure drop correlation showed the deviation of 30% for R-446A and 40% for R-1234ze(E).

Keywords: evaporation, heat transfer coefficient, pressure drop, R-446A, R-1234ze(E)

1. Introduction

HFC-410A is about to be replaced with low GWP refrigerant such as HFO-446A, HFO-447A due to its high GWP, even though it has been used for the last couple of decades as HCFC-22 substitute for the refrigeration and air-conditioning system.

HFC-134a is about to be replaced with HFO-1234ze(E) with low GWP for the turbo chiller, even though it has been replaced with HFO-1234yf with low GWP for the automobile air-conditioner. HFO refrigerants such as HFO-446A and HFO-1234ze(E) have been investigated as the alternative refrigerants with low GWP.

The present study measured evaporation heat transfer coefficients and pressure drop characteristics of them that are essential to design the evaporator.

2. Experimental Apparatus

The experimental apparatus is schematically shown in Fig. 1. The test apparatus consisted of a test section, a condenser, a refrigerant pump, a receiver, a mass flow meter, a preheater, and measuring devices. The test section was a horizontal stainless steel tube with an inner diameter of 7.4mm and the length of 1300mm. Outside wall temperature of the tube was measured along the tube by 42 T type thermocouples. Refrigerant pressure was measured by absolute pressure gauge at the inlet, outlet, and three middle points with the same interval of the evaporator.

Flow pattern of the refrigerant was observed by a sight glass installed at inlet and outlet of the test section. The preheater controlled the inlet vapour quality of the refrigerant by heating the refrigerant in front of the test section. Refrigerant passed through the test section was condensed by the condenser and then flowed into the receiver. The refrigerant pump circulated the refrigerant.



Fig. 1: Experimental Apparatus.

Heat transfer coefficient was calculated by using the following equations.

$$q = hA(T_{wi} - T_{sat})$$
(1)

$$T_{wi} = T_{wo} - \frac{t\pi \overline{r_{wi}}}{2\pi kL} \cdot q \tag{2}$$

3. Result and discussion

3.1. Heat transfer coefficient

Figure 2 shows the effect of mass flux and heat flux on the heat transfer coefficient of R-446A.





The heat transfer coefficients increased with increasing the refrigerant mass flux. For the high mass flux condition, nucleate boiling suppression appeared earlier. This means that convective heat transfer appeared earlier under the high mass flux condition. Nucleate boiling is dominant at the initial stage of evaporation. The large effect of heat flux on the heat transfer coefficient shows the dominance of nucleate boiling heat transfer contribution. At the high quality region, nucleate boiling was suppressed, and the effect of heat flux on the heat transfer coefficient was getting lower.

Figure 3 shows the effect of mass flux and heat flux on the heat transfer coefficient of R-1234ze(E). The effect of heat flux and mass flux on the heat transfer coefficient was similar with that of R-446A.



Fig. 3: Effect of mass flux and heat flux on the heat transfer coefficients of R-1234ze(E).

Figure 4 shows the comparison of heat transfer coefficient of R-446A and R-410A. Heat transfer coefficients of R-446A were higher by 16% than those of R-410A. Since the liquid and vapour densities of R410A were larger than those of R-446A, fluid velocity of R-446A is higher than that of R-410A under constant mass flux condition. Dominance of convective boiling for R-446A was appeared at a low quality.



Fig. 4: Comparison of heat transfer coefficients of R-410A and R-446A.

Typical properties of Refrigerants are shown in Table 1.

Property	R-410A	R-446A	R-134a	R-1234ze(E)
Saturation pressure	936.23kPa	854.28kPa	349.66kPa	259.34kPa
Liquid density(kg/m ³)	1149.6	1058	1278.1	1225.4
Vapour density(kg/m ³)	35.219	25.089	17.131	13.923
Liquid viscosity(μ Pa · s)	151.86	153.47	250.11	253
Vapour viscosity(μ Pa · s)	12.462	11.795	10.911	11.398
Surface tension(mN/m)	8.2984	11.039	10.73	11.497

Table 1: Properties of Refrigerants at Tsat of 5°C.

Correlation by wojtan et al.^[1] predicted the heat transfer coefficients of R-446A with the deviation of 45%, and the heat transfer coefficients of R-410A lower by 20% than the measured data.

Figure 5 shows the comparison of heat transfer coefficient of R-134a and R-1234ze(E). Heat transfer coefficients of R-134a were slightly higher than those of R-1234ze(E). Since liquid and vapour densities of R-134a are large, fluid velocity of R-1234ze(E) was higher than that of R-134a at constant mass flux. It causes dominance of convective boiling and large heat transfer coefficient. Pressure of R-134a is generally higher than that of R-1234ze(E). At large pressure, bubble frequency increased with the decrease of bubble departure diameter. It contributed to the increase of the heat transfer coefficient. The heat transfer coefficients of R-134a were slightly higher than those of R-1234ze(E)



Fig. 5: Comparison of heat transfer coefficients of R-134a and R-1234ze(E).

Correlation by Choi et al.^[2] predicted the heat transfer coefficients of R-1234ze(E) with the deviation of 30%, and the heat transfer coefficients of R-134a lower by 20% than the measured data.

3.2. Pressure drop

Figure 6 shows the comparison of pressure drop of R-410A and R-446A. Pressure drops of R-446A were larger by approximately 30% than those of R-410A. Since the density ratio of liquid and vapour of R-446A is higher than that of R-410A, interfacial shear stress of R-446A is higher than that of R-410A.



Fig. 6: Comparison of Pressure drop of R-410A and R-446A.

Pressure drop predicted by the Friedel^[3] correlation showed the deviation of 30% from the measured data for R-446A. Figure 7 shows the comparison of pressure drop of R-134a and R-1234ze(E). Pressure drops of R-1234ze(E) were larger by approximately 20% than those of R-134a. Since the density ratio between liquid and vapour of R-1234ze(E) is higher than that of R-134a, interfacial shear stress of R-1234ze(E) is higher than that of R-134a. Pressure drop predicted by the Friedel^[3] correlation showed the deviation of 40% from the measured data for R-1234ze(E).



Fig. 7: Comparison of Pressure drop of R-134a and R-1234ze(E).

4. Conclusions

The heat transfer coefficients of R-446A were larger by approximately 16% than those of R-410A, while The heat transfer coefficients of R-1234ze(E) were lower by 3% than those of R134a. Pressure drops of R-446A were larger by approximately 30% than those of R-410A, while the pressure drops of R-1234ze(E) were larger by 20% than those of R134a.

Wojtan et al.^[1] correlation predicted the heat transfer coefficients of R-446A with the deviation of 45%, while Choi et al.^[2] correlation predicted the heat transfer coefficients of R-1234ze(E) with the deviation of 30%. Pressure drop correlation predicted by Friedel^[3] correlation showed the deviation of 30% for R-446A and 40% for R-1234ze(E).

Acknowledgements

This work was supported by the Technology Innovation Program(10052926, Development of Core Technologies for Low GWP Refrigerant System) funded by the Ministry of Trade, Industry&Energy(MI, Korea).

References

- [1] L. Wojtan, T. Ursenbacher and J. R. Thome., "Investigation of flow boiling in horizontal tubes: Part 1- A new diabatic two-phase flow pattern map," *Int. J. Heat and Mass Transfer*, vol. 48, pp. 2955-2969, 2005.
- [2] T. Y. Choi, Y. J. Kim, M. S. Kim, and S. T. Ro., "Evaporation heat transfer of R-32, R-134a, R-32/134a, and R-32/125/134a inside a horizontal smooth tube," *Int. J. Heat and Mass Transfer*, vol. 43, pp. 3651-3660, 2000.
- [3] L. Friedel, "Improved friction pressure drop correlations for horizontal and vertical two-phase pipe flow," *European Two-Phase Flow Group Meeting*, Italy, Ispra, 1979, Paper E2.