

CFD Simulations of Gas Holdup in a Bubble Column at High Gas Temperature of a Helium-Water System

Mohammed W. Abdulrahman
Rochester Institute of Technology
Dubai, UAE
mwacad@rit.edu

Abstract - Bubble columns (BCs) are systems that contain two-phases; gas, and liquid, in which gaseous bubbles are dispersed through a liquid in a vertical column. They have a wide range of applications because of their many advantages. The hydrodynamics of the BCs have a significant effect on its scale up analysis. The most important parameter that can be used for describing the performance of the BCs is the gas holdup. In this paper, the overall gas holdup was predicted by developing computational fluid dynamics (CFD) simulations for a helium-water bubble column, where helium gas is injected at 90°C through a liquid of water at 22°C. The approaches used to model the bubble column by CFD is 2D plane. From the CFD simulations, it was found that the overall gas holdup increases by increasing the superficial gas velocity at any specific static liquid height. In addition, the overall gas holdup decreases by increasing the static liquid height at any given superficial gas velocity. All CFD simulation results of gas holdup were validated by experimental results of helium-water BCs from previous literature with good agreement, which demonstrates the applicability of the simulations for this kind of BC systems. The CFD results were validated for superficial gas velocities up to 0.15 m/s, and aspect ratios up to 4. From the comparison with the experimental results, it was found that in general, the profiles of gas holdup determined from CFD simulations, under-predicted the experimental data. Also, it was shown that the experimental effects of the static liquid height on gas holdup were correctly predicted by CFD simulations. Moreover, it was observed that the distribution of gas holdup along the cross-section of the column is unequal, where the gas holdup is higher at the center of the column and lower near the wall region.

Keywords: Bubble column reactors; CFD; Gas holdup; Multi-phase; Hydrodynamics

1. Introduction

Bubble columns are used in different industrial applications because of their advantages that motivates doing more detailed studies with this type of reactors. The advantages are; better temperature control; lower pressure drop; and high volumetric heat transfer rates. Additional advantages include; higher values of effective interfacial areas; relatively cheap to construct and operate, require less floor space [1]; and no challenges of material selection for heating jacket or insertions. In spite of the fact that BCs are simple in construction, accurate scale-up of such reactors requires a comprehensive knowledge of the hydrodynamic and heat transfer characteristics at the same conditions of the targeted process [2]. This scale-up generally depends on the evaluation of heat transfer and mixing characteristics, as well as chemical kinetics of the reacting system [3]. The hydrodynamics of the BC have a significant effect on its scale up analysis. In BCs, gas phase that is moving upward transfers momentum to the liquid phase that is either stagnant or moving slower than the gas. Therefore, the hydrodynamics of BCs are controlled mainly by the gas flow [2]. It has been reported that the operating conditions and design as well as the geometry of the column strongly affect the hydrodynamics of the BCs [4-6]. One of the most important hydrodynamic characteristics of the BCs is the gas holdup.

In the literature, there are many studies related to BCs, such as hydrodynamics and flow regimes, as well as design and scale up analyses. Nigar et al. [3] have reviewed bubble column reactors (BCRs) by focusing on the reactor design, fluid dynamics and flow regime transitions of reactors. The most investigated parameter was gas holdup and for gas velocities up to 35 cm/s [3]. Behkish [7] has used two BCRs with different diameters to measure the gas holdup for five different gases (N₂, H₂, CO, He and CH₄) in Isopar-M liquid [7]. The data were obtained under wide ranges of pressures (1-27 bar), superficial gas velocities (0.08-0.4 m/s), and temperatures (323-453K). Abdulrahman [8-12] has studied experimentally the hydrodynamics and direct contact heat transfer in a SBC with a high temperature helium gas and a slurry of liquid water and alumina solid particles. He has formulated new empirical equations for the gas holdup [8, 12] and the volumetric heat

transfer coefficient [10-12] in terms of the Reynolds number, reactor dimensions and solid concentration, for both bubbly and churn-turbulent flow regimes.

The modelling by CFD simulation in multiphase flow has been studied by many researchers [13-18]. In the literature, there is no universal agreement that indicates that the CFD models are capable of predicting the experimental results of multiphase flow regimes [18]. For instance, Delnoij et al. [19, 21] have used Eulerian-Lagrangian approach in a flat bubble column to model two-phase flow by using laminar flow model, as well as drag, lift, virtual-mass, and hydrodynamic-interaction forces [22]. Sokolichin and Eigenberger [23] have obtained same results by using finer grid size and neglecting the effects of virtual-mass and lift forces, as well as bubble-bubble interactions. Deen et al. [24] have found that using the virtual mass force will not influence the results. Krishna and Van Baten [25] have studied high pressure turbulent flow simulations and took into consideration the drag force only. They have found that there is high uncertainty when adding the effects of lift forces of small and large bubbles. Moreover, they have noted that there is no effect of the virtual mass force on the results of the simulations. Abdulrahman [26, 27] has performed CFD analyses to study the volumetric heat transfer coefficient and the temperature distribution in a direct contact heat transfer for a helium-water-alumina slurry bubble column reactor, where helium gas is injected at 90°C through a slurry of water at 22°C and alumina solid particles. He has studied the effects of superficial gas velocity, static liquid height, and solid particles concentration, on the volumetric heat transfer coefficient and the temperature distribution of the SBC. The results of CFD simulations were compared with experimental data from previous literature and show that the profiles of the volumetric heat transfer coefficient calculated from CFD models, generally under-predicts the experimental data. The CFD model correctly predicted the experimental effects of static liquid height and solid concentration on volumetric heat transfer coefficient.

Law et al. [28] have determined the average gas holdup by using 2-D and 3-D models for a bubble column. They have examined the influence of various cell resolutions in calculating the average gas holdup by using FLUENT. They have found that there is a good agreement between the 2-D simulations and the experiments of Rampure et al. [29] with a cell size of 0.67 cm. Also, they have reported that if the sizes of the cells are smaller than the size of the bubble, this will lead to unreasonable results. Moreover, they have found that both 3-D and 2-D simulations will predict the same results if their resolutions are comparable. Rampure et al. [29] have investigated the setup of FLUENT by carrying out experiments on a cylindrical column with two and three phase systems to measure gas holdup. They have found that the experimental results were lower than the CFD results and the agreement of the results was acceptable [30]. Krishna et al. [25] have investigated experimentally gas holdups for small and large bubbles with different systems and different column diameters. Also, they have determined the overall gas holdup by using CFD in an Eulerian framework and compared the CFD results with the experimental results for three different column diameters. They have reported that the results were in a good agreement [2].

In this paper, the objective is to study the CFD analyses of the BC from the perspective of hydrodynamics in a steady state condition. The purpose of the numerical works is to develop and validate new predictive CFD models by using the well-known software packages (ANSYS FLUENT). The CFD models will be developed by using two dimensional simulations to predict the values of the gas hold-up at different superficial gas velocities, and static liquid heights.

2. CFD Simulations of the Multiphase Flow

As indicated by Abdulrahman [26], the CFD simulations in this paper are performed for a 2D plane and a steady state system with Eulerian-Eulerian model, Eulerian sub-model, and pressure based solver type. The equations used in the CFD analysis in this paper are shown in details in Table 1.

Table 1: Details of equations used in the CFD simulations.

Description [reference]	Phase	Equation	Notes
Volume equation [31]	Gas	$V_g = \int_V \alpha_g dV$	Equations of V_g and V_l

	Liquid	$V_l = \int_V \alpha_l dV$	must satisfy: $\alpha_g + \alpha_l = 1$
Continuity equation in 2D Cartesian coordinates (x, y) [32]	Gas	$\frac{\partial v_{x,g}}{\partial x} + \frac{\partial v_{y,g}}{\partial y} = 0$	
	Liquid	$\frac{\partial v_{x,l}}{\partial x} + \frac{\partial v_{y,l}}{\partial y} = 0$	
Momentum equation in 2D Cartesian coordinates [33]	Gas <u>x - direction</u>	$\rho_g \alpha_g \left(\frac{\partial v_x}{\partial t} + v_x \frac{\partial v_x}{\partial x} + v_y \frac{\partial v_x}{\partial y} \right) = -\alpha_g \frac{\partial P}{\partial x} + \alpha_g \frac{\mu_{g,eff}}{3} \frac{\partial(\nabla \cdot \mathbf{V})}{\partial x} + \mu_{g,eff} \alpha_g \left[\frac{\partial^2 v_x}{\partial x^2} + \frac{\partial^2 v_x}{\partial y^2} \right] + \rho_g \alpha_g g_x + M_{i,g,x}$	
	Gas <u>y - direction</u>	$\rho_g \alpha_g \left(\frac{\partial v_y}{\partial t} + v_x \frac{\partial v_y}{\partial x} + v_y \frac{\partial v_y}{\partial y} \right) = -\alpha_g \frac{\partial P}{\partial y} + \alpha_g \frac{\mu_{g,eff}}{3} \frac{\partial(\nabla \cdot \mathbf{V})}{\partial y} + \mu_{g,eff} \alpha_g \left[\frac{\partial^2 v_y}{\partial x^2} + \frac{\partial^2 v_y}{\partial y^2} \right] + \rho_g \alpha_g g_y + M_{i,g,y}$	
	Liquid <u>x - direction</u>	$\rho_l \alpha_l \left(\frac{\partial v_x}{\partial t} + v_x \frac{\partial v_x}{\partial x} + v_y \frac{\partial v_x}{\partial y} \right) = -\alpha_l \frac{\partial P}{\partial x} + \alpha_l \frac{\mu_{l,eff}}{3} \frac{\partial(\nabla \cdot \mathbf{V})}{\partial x} + \mu_{l,eff} \alpha_l \left[\frac{\partial^2 v_x}{\partial x^2} + \frac{\partial^2 v_x}{\partial y^2} \right] + \rho_l \alpha_l g_x + M_{i,l,x}$	
	Liquid <u>y - direction</u>	$\rho_l \alpha_l \left(\frac{\partial v_y}{\partial t} + v_x \frac{\partial v_y}{\partial x} + v_y \frac{\partial v_y}{\partial y} \right) = -\alpha_l \frac{\partial P}{\partial y} + \alpha_l \frac{\mu_{l,eff}}{3} \frac{\partial(\nabla \cdot \mathbf{V})}{\partial y} + \mu_{l,eff} \alpha_l \left[\frac{\partial^2 v_y}{\partial x^2} + \frac{\partial^2 v_y}{\partial y^2} \right] + \rho_l \alpha_l g_y + M_{i,l,y}$	
Effective density	Gas	$\hat{\rho}_g = \alpha_g \rho_g$	
	Liquid	$\hat{\rho}_l = \alpha_l \rho_l$	
Effective dynamic viscosity [22, 34]	Gas	$\hat{\mu}_g = \frac{\rho_g}{\rho_l} \hat{\mu}_l$	For $k - \epsilon$ turbulence model
	Liquid	$\hat{\mu}_l = \mu_{lam,l} + \mu_{tur,l} + \mu_{BIT,l}$	
Total interfacial force acting between the phases [31]		$M_{i,l} = -M_{i,g} = M_D$	It is obtained by neglecting the lift force [18] and the virtual mass force [36]
Drag force [31]		$M_D = \frac{\rho_g f}{6 \tau_b} d_b A_i (\mathbf{V}_g - \mathbf{V}_l)$	For gas-liquid system
Interfacial area [31]		$A_i = \frac{6 \alpha_g (1 - \alpha_g)}{d_b}$	
Particular relaxation time [31]		$\tau_b = \frac{\rho_g d_b^2}{18 \mu_l}$	
Drag function [31]		$f = \frac{C_D Re}{24}$	
Reynolds number [31]		$Re = \frac{\rho_l \mathbf{V}_g - \mathbf{V}_l d_b}{\mu_l}$	$ \mathbf{V}_g - \mathbf{V}_l $: slip velocity d_b : Sauter-mean diameter
Schiller-Naumann drag equation [36]		$C_D = \begin{cases} \frac{24 (1 + 0.15 Re_b^{0.687})}{Re_b} & Re_b \leq 1000 \\ 0.44 & Re_b > 1000 \end{cases}$	

In the modelling of turbulence, the model that is usually used because of its accuracy is the standard $k - \varepsilon$ model. It solves two transport equations separately in which the turbulent velocity and length scales are determined independently. In the derivation of the standard $k - \varepsilon$ model, the molecular viscosity was neglected and the flow was assumed to be fully turbulent [31]. On the other hand, $k - \omega$ model shows better performance for low Reynolds number flows, but the negative aspect of this model is its relatively strong sensitivity of the solution to the values of k and ω outside the shear layer (free stream sensitivity). For this reason, the use of the standard $k - \omega$ model is not generally recommended in ANSYS FLUENT [31]. For multiphase systems with high density ratio between gas and liquid phases such the system of this paper, two sub models can be used in the standard $k - \varepsilon$ turbulence model: the dispersed turbulence model, and a per-phase turbulence model [31]. In this paper, the dispersed turbulence model is used to model the turbulence in the bubble column, because of its less computational efforts [31]. The wall functions that have been most widely used in industrial flows, are the standard wall functions [31]. They are reasonably accurate for most of high-Reynolds-number flows with wall bound. In standard wall functions, the meshes created near the wall must be very fine to achieve the value of y^+ within the log-law layer of the boundary layer (i.e. $30 \leq y^+ \leq 300$) [26]. In this paper, the standard wall functions are used.

In this paper, the 2D-CFD analysis of the BC is studied using the ANSYS FLUENT V.13 software. The overall diameter of the BC is taken to be 21.6 cm and height of static liquid of the BC is taken to have three different values (45, 55, and 65 cm). First, ANSYS WORKBENCH V.13 was implemented to draw 2D geometries of the BC and to create meshing. Quadratic mapped mesh is used for the area of the BC and a very fine mesh is used near the wall. The size of the mesh is selected so that there is no dependence of the gas holdup (α_g) on grid. After investigating the grids for different H and U_{gs} , it has been found that the most unfavourable situation is for $H = 65 \text{ cm}$ and $U_{gs} = 0.15 \text{ m/s}$. Table 2 shows the grid independence study that was used to select the optimum grid distribution of the BC problem. From Table 2, it can be seen that the optimum grid is when the number of cells is 20,203 cells, because this will provide minimum relative errors of 0.43% for the values of α_g , when compared with the grid of 56,341 cells. When using the grid of 56,341 cells, the memory requirement of the computer as well as the calculation time will increase significantly because the number of cells is more than twice of that of 20,203. Since the relative errors of α_g between the two grids are very small, it is preferred to use the grid of 20,203 cells and reduce the memory requirement of the computer and the calculation time. The quantity of interest that is monitored during the CFD simulation is the overall gas holdup. The convergence criteria of the simulation is to ensure that the quantity of interest reached a steady state simulation and the residual RMS error values were less than 10^{-4} . Table 3 summarizes the setup of the BC problem in ANSYS FLUENT.

Table 2: Grid independence test for a helium-water BC ($H = 65 \text{ cm}$, $U_{gs} = 0.15 \text{ m/s}$).

Total Cells	3,704	20,203	34,288	56,341
α_g (%)	22.3	23	22.8	23.1

Table 3: Summary of the BC problem setup in ANSYS FLUENT.

General	<i>Solver Type</i>	Pressure-Based
	<i>Velocity Formulation</i>	Absolute
	<i>Time</i>	Steady
	<i>Gravity</i>	ON
	<i>2D Space</i>	Planar
Models	Multiphase-Eulerian	
	Energy-On	
	Viscous-Standard $k - \varepsilon$, Standard Wall Function, Dispersed	
Materials	Water liquid	

	Helium gas			
Phases	Primary phase=liquid phase			
	Secondary Phase=gas phase			
Bubble Diameter	Sauter-mean diameter			
Solution Methods	<i>Scheme</i>	Phase-Coupled SIMPLE		
	<i>Spatial Discretization</i>	Gradient	Least Squares Cell Based	
		Momentum	Second Order Upwind	
		Volume Fraction	First Order Upwind	
		Turbulent Kinetic Energy	Second Order Upwind	
		Turbulent Dissipation Rate	Second Order Upwind	
		Energy	Second Order Upwind	
		Interfacial Area Concentration	Second Order Upwind	
Number of Iterations	100,000			

The boundary conditions of the BC can be represented by inlet, outlet and wall boundary conditions. The inlet volume fraction of the gas is equal to 1 and the inlet velocity of the gas is considered uniform and equal to the volumetric flow rate of the gas divided by the total cross-sectional area of the sparger's orifices. According to Akhtar et al. (2006), the pressure boundary condition can be applied at the outlet of the column because it will produce better convergence. In all simulations, the outlet pressure is equal to atmospheric pressure. The no-slip boundary conditions are applied at the walls of the BC. Symmetry conditions were not used in the simulations to be able to obtain better behaviors of hydrodynamics. Because of the estimation difficulty of the liquid turbulence at the inlet and outlet boundary conditions of the liquid phase, iterations were used to specify the turbulent kinetic energy and dissipation rate.

3. Results

In this section, the CFD models results of the hydrodynamic studies for steady-state conditions are introduced. The effects of superficial gas velocities (U_{gs}), and static liquid heights (H) on gas holdup are investigated.

3.1. Effects of Superficial Gas Velocity (U_{gs}) and Static Liquid Height (H) on α_g

Figures 1-3 show the effect of superficial gas velocity (U_{gs}) and static liquid height (H), on the numerical overall gas holdup (α_g) of the helium-water BC. From these figures, it can be seen that α_g increases by increasing U_{gs} and/or decreasing H . Figs. 4 and 5 show the effect of U_{gs} and H on the contours of α_g . It can be seen in these figures that the radial distribution of the gas holdup is unequal, where the gas holdup is high in the center and low in the wall region. This behavior is due to the gradient of buoyancy forces between the column center and the wall region as indicated by Shaikh and Al-Dahhan [37].

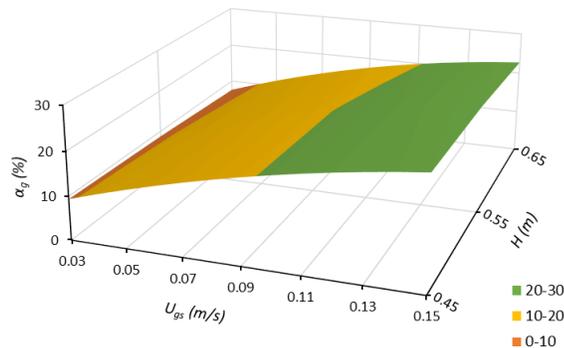


Fig. 1: Numerical overall gas holdup versus U_{gs} and H of helium-water BC.

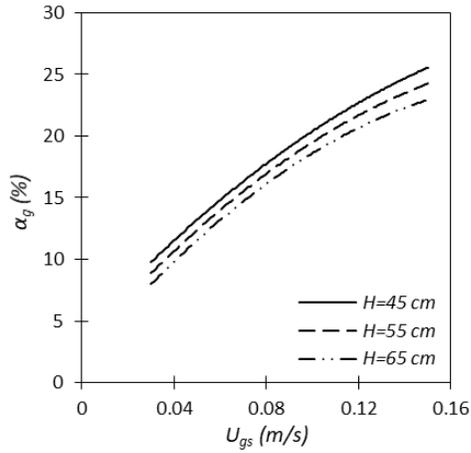


Fig. 2: Effect of static liquid height on numerical α_g versus U_{gs} of helium-water BC.

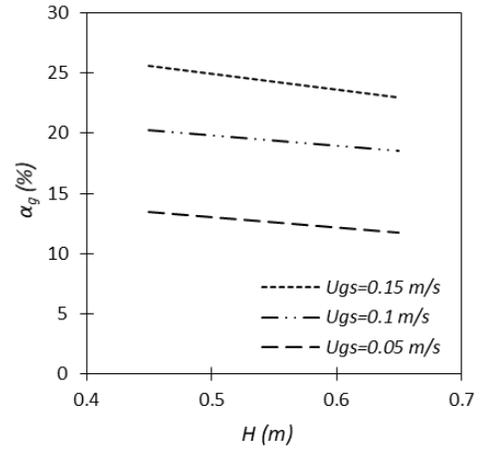


Fig. 3: Numerical overall gas holdup versus H of helium-water BC for different U_{gs} .

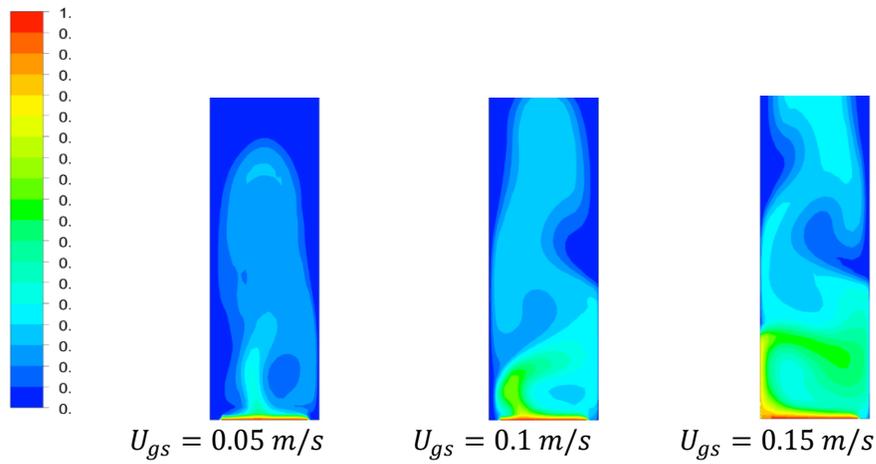


Fig. 4: Contours of gas holdup of a helium-water BC, $H = 65$ cm and different U_{gs} .

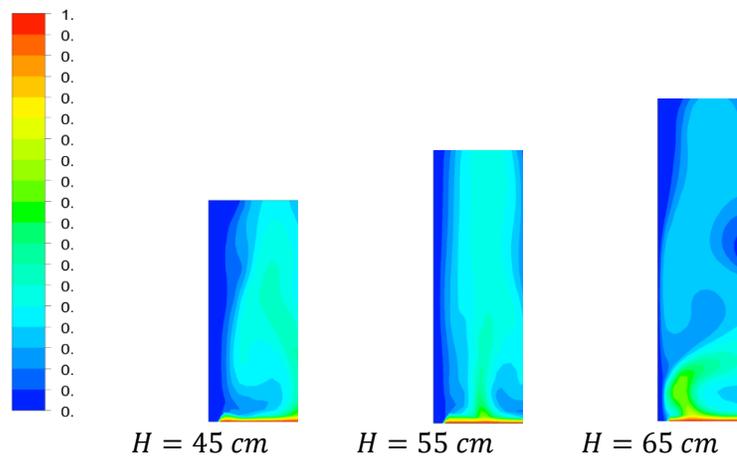


Fig. 5: Contours of α_g of a helium-water BC, $U_{gs} = 0.1$ m/s and different H .

3.2. Comparison of Numerical α_g with the Experimental Data

To validate the numerical (CFD) data of hydrodynamic studies produced in the helium-water BC, a comparison is made with the experimental data of Abdulrahman [8, 12]. Fig. 6 compares between the CFD simulations and experimental results of α_g versus U_{gs} for different H . In general, it can be seen that all profiles of α_g calculated from CFD models under-predict the experimental data with a maximum relative error of less than 28.5%. Considering the complexity of the multi-phase flow in bubble columns, the agreement is satisfactory and encouraging. The reduction in the results of CFD models is caused by the use of a 2D-plane mesh producing lower gas flow rates when compared with the 3D column. Also, the CFD model applied the source for the gas phase across the base of the column, ignoring the effect of the sparger height and therefore over-estimating the static liquid height (H). Due to that, the overall gas holdup is under-estimated when compared with the experimental flows. Another reason for the reduction of α_g is that the turbulent nature of the flow demands the use of a very fine mesh to realize all the vortical structures in the flow, especially for the smaller eddies [23]. The ability of the CFD model to account for H effect on α_g versus U_{gs} is also assessed by comparison to the experimental data of Abdulrahman [8, 12] as shown in Figs. 7 and 8. Fig. 7 shows that, the curves of α_g versus U_{gs} at different values of H , are approximately parallel to each other, which means that the values of α_g versus U_{gs} decreases almost with a constant value by increasing H . In other words, the rate of decrease of α_g versus H is higher at lower U_{gs} . This behavior of α_g versus U_{gs} is also shown in Fig. 8 with α_g versus H , where the curves are approximately parallel to each other for different values of U_{gs} . The above experimental behaviors of α_g versus U_{gs} and H are correctly predicted by the CFD model.

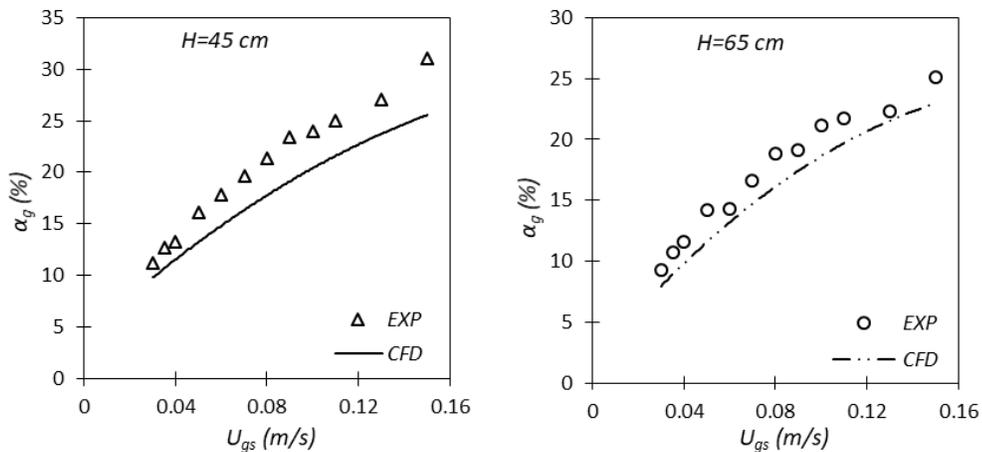


Fig. 6: Comparison of α_g versus U_{gs} between CFD and experimental results for different H of helium-water BC.

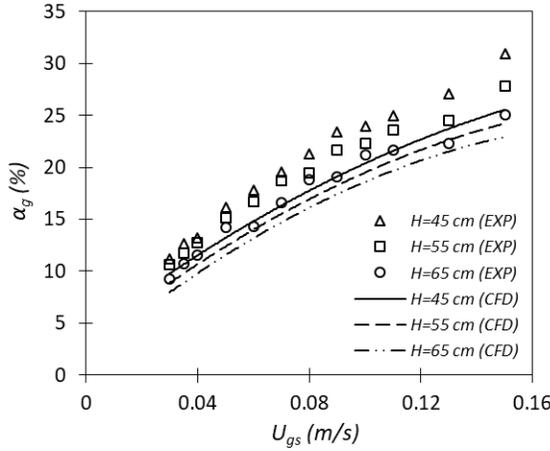


Fig. 7: Effect of H on α_g versus U_{gs} of helium-water BC for experimental data and CFD model.

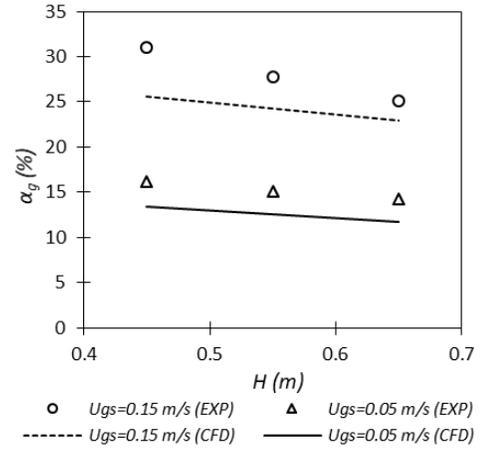


Fig. 8: Comparison between CFD and experimental α_g versus H of helium-water BC for different U_{gs} .

4. Conclusion

Two dimensional CFD simulations of helium-water system were developed to predict the values of the gas hold-up at different superficial gas velocities, and static liquid heights. In this paper, the multiphase Euler-Euler method was used for the numerical solutions and the standard $k - \varepsilon$ dispersed turbulence model was used for modeling the turbulence in the bubble column. The data of gas holdup were validated against the experimental data and showed good agreement. The validation of the CFD simulations with the experimental data demonstrates the applicability of the simulations for the helium-water bubble column systems. The CFD simulations were validated for superficial gas velocities up to 0.15 m/s, and aspect ratios up to 4. From the CFD simulations, the following points are concluded;

- In general, the profiles of α_g determined from CFD simulations, under-predicted the experimental data.
- The experimental effect of H on α_g was correctly predicted by CFD simulations.
- The distribution of gas holdup along the cross-section of the column is unequal, where the gas holdup is higher at the center of the column and lower near the wall region.

5. Nomenclature

Symbol	Definition	Symbol	Definition
A_i	Interfacial area (m^2)	\mathbf{V}_g	Velocity field of gas phase (m/s)
C_D	Drag coefficient	\mathbf{V}_l	Velocity field of slurry phase (m/s)
d_b	Bubble diameter (m)	α_g	Gas holdup
f	Drag function	α_l	Liquid holdup
g	Gravitational acceleration (m^2/s)	μ_{eff}	Effective dynamic viscosity (Pa.s)
H	Height of static liquid (m)	μ_g	Dynamic viscosity of gas phase (Pa.s)
M_D	Drag force (N/m^3)	μ_l	Dynamic viscosity of liquid phase (Pa.s)
M_i	Total interfacial force acting between phases (N/m^3)	$\hat{\mu}_g$	Effective dynamic viscosity of phase (Pa.s)
P	Pressure (Pa)	$\hat{\mu}_l$	Effective dynamic viscosity of liquid (Pa.s)
Re	Reynolds number	$\mu_{lam,l}$	Molecular viscosity (Pa.s)
U_{gs}	Superficial velocity of gas (m/s)	$\mu_{tur,l}$	Shear-induced turbulent viscosity (Pa.s)
$v_{x,g}$	Velocity component in x -direction of gas phase (m/s)	$\mu_{BIT,l}$	Bubble-induced turbulence viscosity (Pa.s)

$v_{x,l}$	Velocity component in x -direction of slurry phase (m/s)	ρ	Density (kg/m ³)
$v_{y,g}$	Velocity component in y -direction of gas phase (m/s)	ρ_g	Density of gas (kg/m ³)
Symbol	Definition	Symbol	Definition
$v_{y,l}$	Velocity component in y -direction of slurry phase (m/s)	ρ_l	Density of liquid (kg/m ³)
V_g	Volume of gas phase (m ³)	$\hat{\rho}_g$	Effective density of gas phase (kg/m ³)
V_l	Volume of liquid phase (m ³)	$\hat{\rho}_l$	Effective density of liquid phase (kg/m ³)
V_s	Volume of solid phase (m ³)	τ_b	Particulate relaxation time (s)

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